PNEUMATIC OIL LUBRICATOR

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BACKGROUND OF THE PRESENT INVENTION

The present invention relates to an oiler and more particularly relates to a pneumatic oil lubricator which can activate or stop the pneumatic oil pump automatically when the oil output, such as an oil gun, is switched on or switched off.

Conventional pneumatic oil lubricators comprise a 10 pneumatic motor and an oil pump. A drive shaft of the motor is driven to rotate by compressed air. The rotating drive shaft can drive the interior vanes of the oil Pump to rotate for pumping oil from an oil reservoir to an outlet oil gun via oil hoses.

Since the oil pump will continue functioning even when the oil gun is switched off to close the outlet, the conventional pneumatic oil lubricator should comprise a pouring-back apparatus with safety valve which enables the oil to flow back to the oil reservoir in order to 20 prevent the oil hoses from over pressured.

Practically, even the outlet oil gun is switched off, the conventional oiler, that is the pneumatic motor and the oil pump, will continue work. It is so boresome and inconvenient for an operator, who is operating the oil 25 gun, to get back to turn off the oiler because it is always a long distance between the oiler and the oil gun. However, if we leave the pneumatic motor and the oil pump to actuate when the outlet is closed, it is properly wasting the power source.

SUMMARY OF THE PRESENT INVENTION

The main object of the present invention is to provide a pneumatic oil lubricator which can activate or stop the pneumatic oil pump automatically without the uti- 35 lizing of electricity or electric equipment when the oil outlet unit, such as an oil gun, is switched on or switched off.

Another object of the present invention is to provide a pneumatic oil lubricator which oil pump is specifically 40 designated for pneumatic oil lubricator that it has smaller size and simplified structure with lower cost than the convention's.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a sectional view of a preferred embodiment according to the present invention.

FIG. 2 is an exploded isometric view of the pneumatic oil pump of the above embodiment according to the present invention.

FIG. 3 is an exploded isometric view of the automatic controlling apparatus of the above embodiment according to the present invention.

FIG. 4 is a perspective view of the above embodiment according to the present invention.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

Please referring to FIGS. 1 and 4, the present invention is a pneumatic oil lubricator which comprises a 60 pneumatic power source 10, an oil reservoir 20, a pneumatic oil pump 30, a safety valve 40, an automatic controlling apparatus 50, a plurality of oil hoses 60, at least an oil outlet unit 70, a stand 80, and an oil inlet unit 90.

pressed air generally is an air compressor.

The pneumatic oil pump 30, as shown in FIGS. 1 and 2, comprises a hollow case which has a first end case 31

and a second end case 32. The second end case 32 is oil-sealed and axially connected with the first end case 31. The first end case 31 provides an annular air groove 301 at the interior wall which has an air outlet 301a. The second end case 32 provides an oil inlet 322 and an oil outlet 323.

The pneumatic oil pump 30 further comprises an air seat 303, a vane spindle 305, a sleeve 324, at least three gears 312, a gear ring 313, a drive shaft 315, an oil seat 320 and two end covers 34 and 35.

The air seat 303, coaxially disposed in a forward end of the first end case 31, has at least an air inlet passage 304 therein. The sleeve 324 is coaxially disposed in the first end case 31 and secured to the air seat 303. The sleeve 324 also provides at least an air hole 3241 thereon and two longitudinal air grooves 3242 on it's inner pe-

The vane spindle 305 comprises a vane seat 306, a first and a second end shafts 307 and 308 which are coaxially connected to the two ends of the vane seat 306 respectively, and a plurality of vanes 309 which are longitudinally mounted on the corresponding longitudinal slots 3051 provided on the outer surface of the vane seat 306. The vane spindle 305 is coaxially disposed in the sleeve 324 and connected to the air seat 303 by means of a pair of bearings 310 which are mounted on the first and second end shafts 307 and 308 respectively. One end of second end shaft 308 forms a gear end portion 311.

The gear ring 313 is coaxially disposed in the first end case 31 and provides an interior near portion 3131. The three gears 312 each has a central hole 314 and is engaged with the interior near portion 3131 of the gear ring 313 and the gear end portion 311 of the second end shaft 308 of the vane spindle 305.

The drive shaft 315 has two sections wherein the first section 316a is disposed in the first end case 31 and the second section 316b is disposed in the second end case 32 by means of a pair of bearings 310b. The first section 316a of the drive shaft 315 comprises at least three drive pins 316 at one end for inserting into the three central holes 314 of the gears 312 respectively. The other end of the first section 316a provides a slot 317. One end of the second section 316b forms a connecting tip 318 which is 45 inserted in the slot 317 for coaxially connecting with the first section 316a. The second section 316b of the frive shaft 315 further comprises a driving head 319 protruded at the other end portion which forms a plurality of outer longitudinal oil grooves 3191.

The oil seat 320, which is coaxially disposed in the second end case 32, has a central opening 321 which provides a plurality of longitudinal inner oil grooves 3211 that the innermost diameter of the central opening 321 is slightly larger than the outermost diameter of the driving head 319. The driving head 319 is disposed in the oil seat 320.

The first end cover 34, which is secured to the forward end of the first end case 31, provides at least an air inlet 302. The second end cover 35 is secured to the rearward end of the second end case 32.

The automatic controlling apparatus 50, as shown in FIGS. 1 and 3, comprises a hollow unit sleeve 501, a piston 507, and a spring 511.

The unit sleeve 501 comprises two portions 5011 and The pneumatic power source 10 which supplies com- 65 5012 which are coaxially screwed together to form an hollow sleeve with a seal 502 therebetween. The unit sleeve 501 has a first and a second connecting inlets 503 and 504 at its two ends respectively. An oil chamber